

密闭取心钻头两相流流场CFD模拟研究

曹龙龙^{1,2,3}, 张恒春^{1,2}, 王稳石^{1,2}, 闫家^{1,2}, 胡晨^{1,3}

(1. 中国地质科学院勘探技术研究所, 河北廊坊 065000;

2. 中国地质调查局深部地质钻探技术研究中心, 河北廊坊 065000;

3. 中国地质大学(北京)工程技术学院, 北京 100083)

摘要: 密闭取心是一种特殊取心作业, 通过密闭取心钻具与密闭取心钻头配合, 在密闭液作用下实现岩心与钻井液隔离, 可获得地层原始含油饱和度及油水动态等数据。密闭钻头设计为双流道结构, 要满足钻井液过流冲洗冷却钻头唇面、携带岩屑要求, 同时密闭液在井底可形成富集区域, 有效保护岩心免受钻井液污染, 密闭取心钻头流道参数是密闭取心钻头设计的关键。本文运用计算流体力学理论, 采用k-epsilon湍流模型及连续性方程对KM型密闭取心钻具配套密闭取心孕镶金刚石钻头的两相流井底流场进行数值模拟, 研究钻头唇面两相流混流状态、流体压力、流速、剪切应力分布特征等, 分析了流场特征对钻头密闭效果、钻进效率和寿命等的影响, 验证了密闭取心孕镶胎块式金刚石钻头流道结构设计参数, 提出了钻头两相流流道优化设计建议。

关键词: 密闭取心; 金刚石取心钻头; CFD模拟; 流场分析; 数值模拟

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CFD simulation of the two phase flow field in the sealed coring bit

CAO Longlong^{1,2,3}, ZHANG Hengchun^{1,2}, WANG Wenshi^{1,2}, YAN Jia^{1,2}, HU Chen^{1,3}

(1. Institute of Exploration Techniques, CAGS, Langfang Hebei 065000, China;

2. Deep Geological Drilling Technology Research Center, China Geological Survey, Langfang Hebei 065000, China;

3. China University of Geosciences, Beijing 100083, China)

Abstract: Sealed coring is a special kind of coring operation. Through the cooperation of the sealed coring tool and the sealed coring bit, the core and drilling fluid can be isolated under the action of sealed fluid, and in-situ oil saturation and oil-water dynamic data can be obtained. The sealed coring bit is designed as a double channel structure, which can meet the requirements of drilling fluid flushing, cooling the bit lip and carrying cuttings. At the same time, the sealed fluid can form an enrichment area at the bottom hole to effectively protect the core from drilling fluid pollution. The channel parameters of the sealed coring bit are the key to the design of the sealed coring bit. Based on the theory of computational fluid dynamics, the k-epsilon turbulence model and continuity equation are used to simulate the two-phase flow field in the bottom hole of KM type sealed coring tool with the sealed coring impregnated diamond bit. The mixed flow state, fluid pressure, flow velocity and shear stress distribution characteristics of the bit lip are studied. The influence of flow characteristics on sealing effect, drilling efficiency and service life of the bit is analyzed, the design parameters of the channel structure of the sealed coring impregnated block diamond bit are verified, and the optimization design suggestions are put forward for the two-phase flow channel of the bit.

Key words: sealed coring; diamond coring bit; CFD simulation; flow field analysis; numerical simulation

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作者简介: 曹龙龙, 男, 汉族, 1989年生, 工程师, 勘查技术与工程专业, 主要从事取心钻探和深部钻探技术研究工作, 河北省廊坊市金光道77号, 13833686142@163.com。

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0 引言

常规取心钻进泥浆通过钻头水眼循环,进入钻头的岩心会直接接触泥浆,受到泥浆中滤液和部分固相组分的污染,使获取的目的地层岩心含油饱和度和地层孔隙度等重要数据失真^[1-2]。密闭取心技术获取的岩心基本不受钻井液污染,能真实再现地层原始地质孔隙度、含油饱和度及水侵和含水率等数据^[3-4]。KM型密闭取心钻具配套密闭取心钻头设计为胎块式孕镶金刚石钻头,钻头唇面设计为双水口,分别过流钻井液和密闭液^[5-6]。应用计算流体力学(CFD),模拟分析钻头在井底正常钻进时钻井液和密闭液两相流过流钻头水口在井底富集状态,以直观的流体分析模拟还原实际钻进时的井底两相流流场特征,为钻头唇面冷却排屑的流体力学特征、密闭液漫流密闭岩心效果评价等提供参考依据^[7-8]。

1 建模及控制方程

1.1 建模及计算条件设定

以KM216型密闭取心钻具配套 $\varnothing 215.9\text{ mm}/\varnothing 124\text{ mm}$ 钻头作为研究模型,该钻头为8个U形胎块式均布圆弧形底唇面,密闭液过流水口为8个直径16 mm的圆形水口,与钻井液的过流水口间隔式排布^[9-10],钻头三维结构和建模如图1和图2所示。

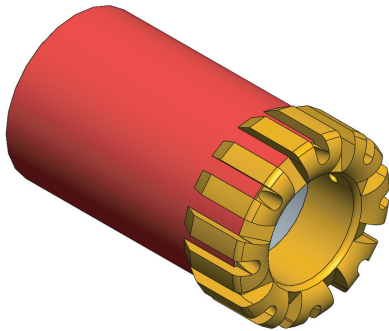


图1 密闭取心钻头三维结构

Fig.1 3D structure of the sealed coring bit

利用三维建模软件构建钻头井底工作时的3D模型计算域,钻头底唇面与井底间距设为0.01 mm,钻头内壁面与岩心环隙间距设为1 mm,钻头外壁面与井壁间距设为2 mm,模型计算域如图3和图4所示。

利用计算流体力学(CFD)分析软件采用四面体网格对计算域进行划分并模拟分析,为简化模拟计算过程,忽略对井底两相流流场影响较小的次要

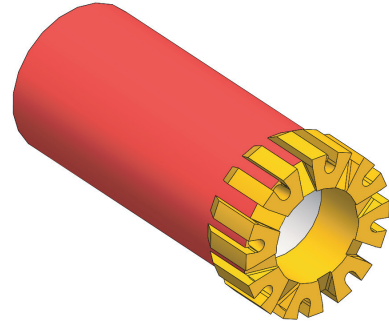


图2 密闭取心钻头三维建模

Fig.2 3D modeling of sealed coring bit

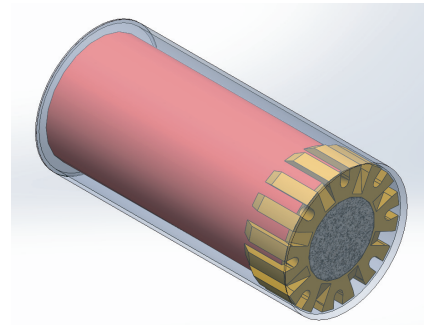


图3 密闭取心钻头井底工作的3D模型计算域

Fig.3 Calculation domain of 3D model for bottom hole operation of the sealed coring bit

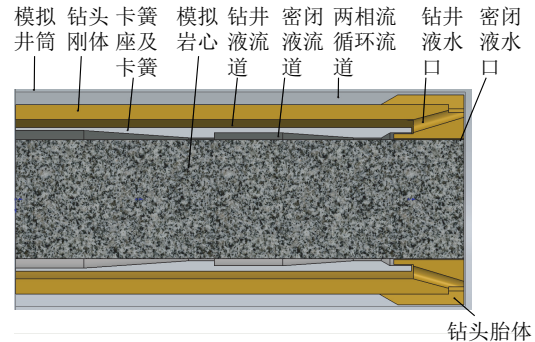


图4 模型计算域剖视图

Fig.4 Section view of the model calculation domain

因素,对边界条件作出以下假设^[11-12]:

- (1)假设井底为平面,井壁为规则的圆柱面,钻头正常钻进时与井底为平面接触;
- (2)忽略两相流流体重力影响因素;
- (3)不考虑岩屑对两相流体的影响。

采用某体系钻井液和某合成基密闭液进行模拟^[13],采用标量混合形式,密度和粘度的变化方式均采用分段线性变化,两相流的入口及出口边界条

件具体设置参数如表1所示。

表1 两相流的具体参数设置

Table 1 Specific parameter setting of the two phase flow

流体类别	密度/ (g· cm ⁻³)	粘度/ (Pa·s)	入口 流量/ (L·s ⁻¹)	出口 压力/ Pa	流态	标 量
某体系钻井液	1.21	0.024	24	0	湍流	0
合成基密闭液	1.1	10	0.005	0	湍流	1

1.2 控制方程及湍流模型

假设钻井液和密闭液进行不可压缩三维非定常湍流流动,建立井底两相流流场的三维湍流物理模型,根据流体动力学基础理论,则流体的连续性方程为:

$$\frac{\partial \rho}{\partial t} + \text{div}(\rho \mathbf{u}) = 0 \quad (1)$$

因钻井液和密闭液两相流介质的流动处于稳态,则

$$\frac{\partial \rho}{\partial t} = 0$$

$$\text{div} \mathbf{u} = \frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} + \frac{\partial u_z}{\partial z}$$

式中: \mathbf{u} ——矢量速度; u_x 、 u_y 、 u_z ——矢量速度 u 在 x 、 y 、 z 轴上的分量; ρ ——密度; t ——时间。

钻井液与密闭液的流动系统遵守动量守恒定律,因此 x 、 y 、 z 三个方向的微分动量守恒方程为:

$$\frac{\partial \rho u_x}{\partial t} + \text{div}(\rho \mathbf{u}_x \mathbf{u}) = \frac{\partial}{\partial x} \left(\mu \frac{\partial u_x}{\partial x} \right) + \frac{\partial}{\partial y} \left(\mu \frac{\partial u_x}{\partial y} \right) + \frac{\partial}{\partial z} \left(\mu \frac{\partial u_x}{\partial z} \right) - \frac{\partial p}{\partial x} + S_x \quad (2)$$

$$\frac{\partial \rho u_y}{\partial t} + \text{div}(\rho \mathbf{u}_y \mathbf{u}) = \frac{\partial}{\partial x} \left(\mu \frac{\partial u_y}{\partial x} \right) + \frac{\partial}{\partial y} \left(\mu \frac{\partial u_y}{\partial y} \right) + \frac{\partial}{\partial z} \left(\mu \frac{\partial u_y}{\partial z} \right) - \frac{\partial p}{\partial y} + S_y \quad (3)$$

$$\frac{\partial \rho u_z}{\partial t} + \text{div}(\rho \mathbf{u}_z \mathbf{u}) = \frac{\partial}{\partial x} \left(\mu \frac{\partial u_z}{\partial x} \right) + \frac{\partial}{\partial y} \left(\mu \frac{\partial u_z}{\partial y} \right) + \frac{\partial}{\partial z} \left(\mu \frac{\partial u_z}{\partial z} \right) - \frac{\partial p}{\partial z} + S_z \quad (4)$$

式中: p ——流体微元体上的压力; S_x 、 S_y 、 S_z ——动量方程广义源项分量; μ ——动力粘度。

但对于湍流,如果直接求解三维瞬态 N-S 方程(2),求解难度较大,工程中广为采用的方法是对瞬态 N-S 方程做时间平均处理,同时补充反映湍

流特性的湍流模型方程,即采用标准的湍流 k - ϵ 模型控制方程。

$$\frac{\partial(\rho k)}{\partial t} + \text{div}(\rho \mathbf{u} k) = \text{div} \left[\left(\mu + \frac{\mu_t}{\sigma_k} \right) \cdot \text{grad} k \right] - \rho \epsilon + \mu_t P_G \quad (5)$$

$$\frac{\partial(\rho \epsilon)}{\partial t} + \text{div}(\rho \mathbf{u} \epsilon) = \text{div} \left[\left(\mu + \frac{\mu_t}{\sigma_\epsilon} \right) \cdot \text{grad} \epsilon \right] - \rho C_2 \frac{\epsilon^2}{k} + \mu_t C_1 \frac{\epsilon}{k} P_G \quad (6)$$

式中: k ——湍动能; ϵ ——湍流耗散率; $\sigma_k = 1.0$; $\sigma_\epsilon = 1.3$; $C_1 = 1.44$; $C_2 = 1.92$; $\mu_t = \rho C_\mu \frac{k^2}{\epsilon}$; $C_\mu = 0.09$ 。

$$P_G = 2 \left[\left(\frac{\partial u_x}{\partial x} \right)^2 + \left(\frac{\partial u_y}{\partial y} \right)^2 + \left(\frac{\partial u_z}{\partial z} \right)^2 \right] + \left(\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right)^2 + \left(\frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right)^2 + \left(\frac{\partial u_y}{\partial z} + \frac{\partial u_z}{\partial y} \right)^2$$

2 网格划分与数值求解

2.1 网格划分

密闭取心钻头为双流道结构,内部装配双卡簧和模拟岩心,既存在圆环形内腔,又有带一定角度的斜过流孔和不规则变截面的环形缝隙,两相流体在钻头底唇面交汇混流形成的流场分布是钻头密闭效果分析的关键。因此,合理的网格划分是提高流场计算准确度的重要前提^[14-15]。密闭取心钻头采用分区划分网格,针对钻头、卡簧、模拟岩心等对模拟结果影响较小的区域采用相对粗网格划分,对于密闭液和钻井液的混合流体域采用网格间隙加密和面加密,并传播变化类型,针对钻头底唇面两相流交汇区域进行多重面加密调整,划分网格后计算节点数为1380360,计算单元数为4023043。网格划分效果如图5所示。

2.2 数值求解

钻井液和密闭液两相流混合流场模拟采用标量混合方式,2种流体的密度变化选用分段线性,根据菲克定律及相关计算公式求得两相流体的扩散系数为 $1.2 \times 10^{-10} \text{ m}^2/\text{s}$,采用 k - ϵ 模型对两相流流场进行数值求解,对流利用改进的 Petrov-Galerkin 有限元算法,流动和标量求解同时运行。

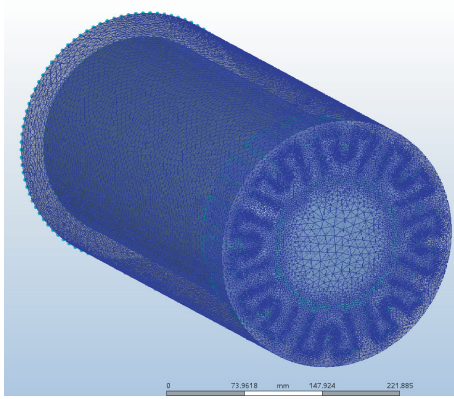


图5 网格划分效果图
Fig.5 Meshing effect

3 模拟结果及分析

3.1 钻头两相流标量混合分析

流体仿真 CFD 软件模拟输出的钻头两相流标量混合流场云图如图 6 所示(红色为密闭液,蓝色为钻井液),密闭液从岩心与钻头内环隙流出后包裹岩心,沿钻头底唇面流道内沿向外均匀辐射,密闭液在钻头底唇面形成的漫流富集区域有效将岩心与钻井液隔离,钻井液从钻头弧形水口流出形成完整的流场域全面覆盖钻头底唇面,冲洗冷却钻头胎体和唇面,保证钻头寿命和钻进效率^[16-17],密闭液与钻井液在钻头底唇面的流场分布及模拟流迹线云图如图 7 所示。

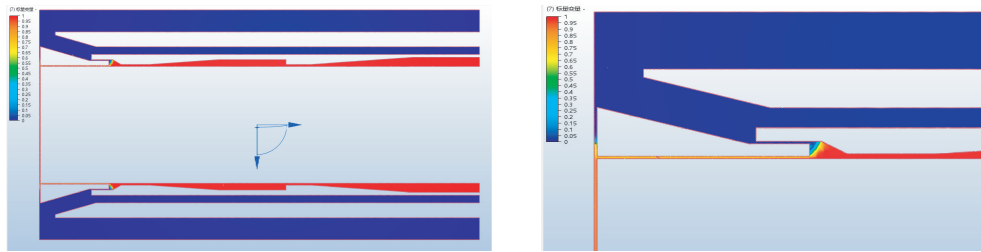


图6 密闭取心钻头两相混流状态云图

Fig.6 Cloud chart of the two phase flow in the mixed flow state of the sealed coring bit

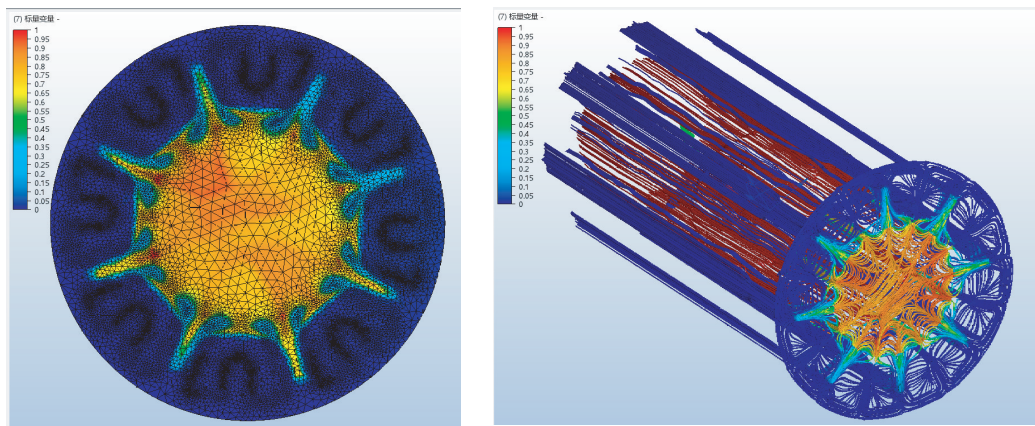


图7 钻头底唇面密闭液与钻井液流场及流迹线

Fig.7 Flow field and trace of the sealing fluid and drilling fluid on the drill bit bottom lip

3.2 井底液流流速及压力分析

密闭钻头两相混合流流速云图如图 8 所示,在钻头斜水口处钻井液流速急剧增大,从位置 2 到 3 流速从 15 m/s 增加到近 28 m/s,钻井液出水口漫流后流速降低,位置 5 处的流速为 10 m/s,钻头底唇面上流速从内沿向外沿逐级降低,但在位置 7 处密闭液与钻井液的交汇区域流体速度很小,接近 0 m/s。钻头底唇面位置 7 所处的环形域流速小,钻进时岩

屑易堆积,但此区域若通过调整钻头水口参数增大流速,钻井液与密闭液对流作用增强,岩心易受泥浆污染,密闭效果变差^[18]。

密闭钻头两相混合流压力云图如图 9 所示,钻井液的流场压力分布梯度从位置 1 到位置 7 逐级降低,在钻头底唇面从位置 4 到位置 5,流场压力从 1 MPa 递减至 0.5 MPa,内沿压力高于外沿压力,有利于形成底唇面的横向漫流冲洗冷却钻头,但在位置

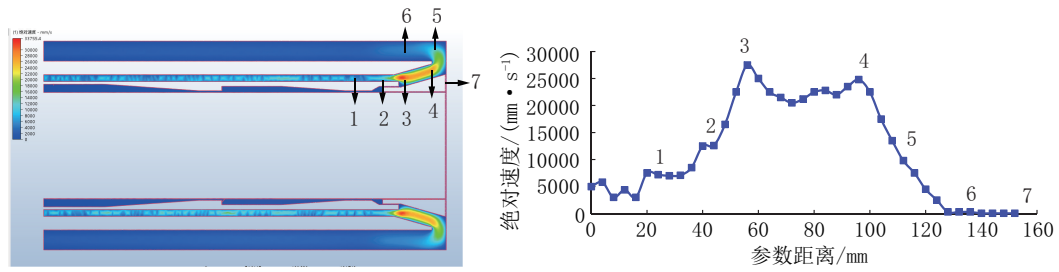


图8 密闭取心钻头钻井液流速云图

Fig.8 Nephogram of the drilling fluid velocity of the sealed coring bit

5和7之间流场压力没有明显的梯度,位置7至5流场区域介质流速低,冲洗冷却效果差,但位置7到5

混流区域压力梯度小,密闭液和钻井液混流速度慢,有利于钻头的密闭效果^[19]。

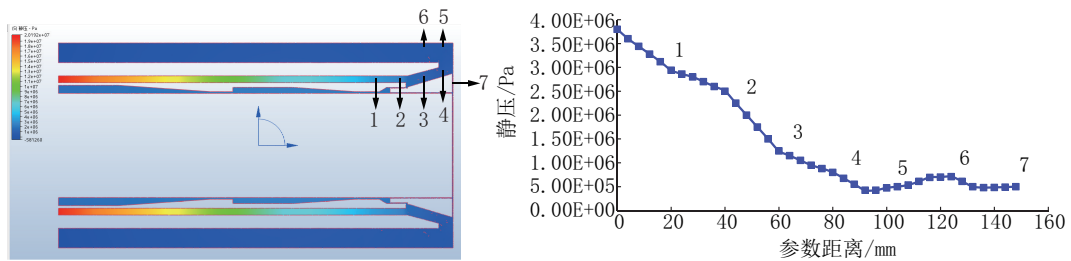


图9 密闭取心钻头流场压力梯度分布

Fig.9 Pressure gradient distribution in the flow field of the sealed coring bit

钻井液与密闭液的两相流介质流速与绝对剪切应力存在一定的比例关系,流体剪力与流体的冲刷效应存在正相关作用^[20],钻头的流体冲刷绝对剪切应力与流速场变化趋势近似契合,如图10所示,底唇面流体剪力从内沿向外沿递减,2个剪力高峰位置3和4处剪力大小为0.05 MPa左右,均避开了密闭液富集保护区域,对密闭效果无影响,钻井液出水口位置4存在强液流冲刷作用,钻头胎体对应部位

会受到冲刷损耗影响,钻头内沿位置7处剪应力低,钻井液侵蚀作用弱,有利于密闭液隔离钻井液密闭岩心效果。钻头底唇面整体剪切应力等值云图如图11所示,在唇面内环形区域与外环形水口周围剪应力分布均匀,无剪切应力集中部位,有利于两相流体在钻头唇面形成稳定流场,提高钻进过程中的岩心密闭率^[21]。

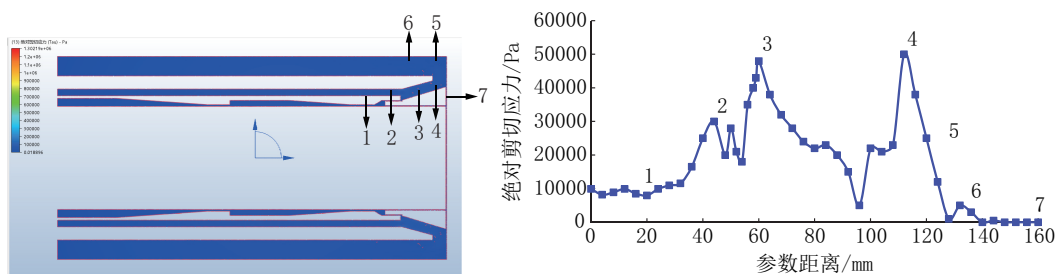


图10 密闭取心钻头流体剪切应力分布

Fig.10 Fluid shear stress distribution of the sealed coring bit

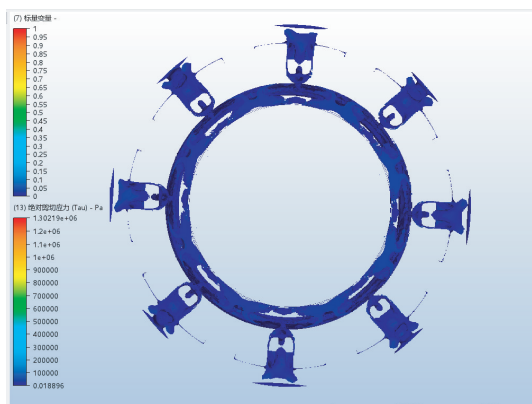


图11 钻头底唇面剪切应力等值云图

Fig.11 Contour nephogram of shear stress on the bit bottom lip

4 结论

(1)通过标量混合方式对密闭金刚石取心钻头密闭液和钻井液两相流流场进行CFD模拟,密闭液与钻井液两相流在钻头底唇面流场分布特征满足密闭取心要求,理论上密闭液可有效隔离钻井液,形成良好的岩心密闭效果,钻头两相流流道及水口结构设计基本合理。

(2)密闭钻头底唇面钻井液与密闭液混流环形区域径向压力梯度小,流体介质流速慢,有利于密闭液形成富集区域密闭岩心,但会影响钻井液在该区域的冲洗冷却效果,长时间钻进中钻头胎体可能会出现局部微小的热损消耗现象。

(3)密闭取心钻头钻井液流道斜角度出水口附近介质流速快,冲刷剪切应力集中,长时间钻进水口附近存在冲刷损耗风险,进而会影响金刚石钻头孕镶热压胎体块二次镶焊强度,因此钻头胎体块水口附近的二次焊接区域应提高焊接精度和强度。

(4)密闭钻头两相流流场模拟得出的结果为后续密闭取心钻头入井试验和进一步性能优化提供了理论依据和方向指导,通过后续的实钻试验与数据反馈,完善模拟边界条件,优化改进钻井液流道倾斜角度和弧形唇面水口参数,减少钻井液冲洗冷却不充分和剪切冲刷应力集中区域,保证岩心密闭率前提下达到提高钻头钻进效率和使用寿命的效果。

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